

# Extending the Operating Range of Brownlee Unit 5

# Brownlee Plant History

- Original Plant built in 1958 with four 100MW rated units

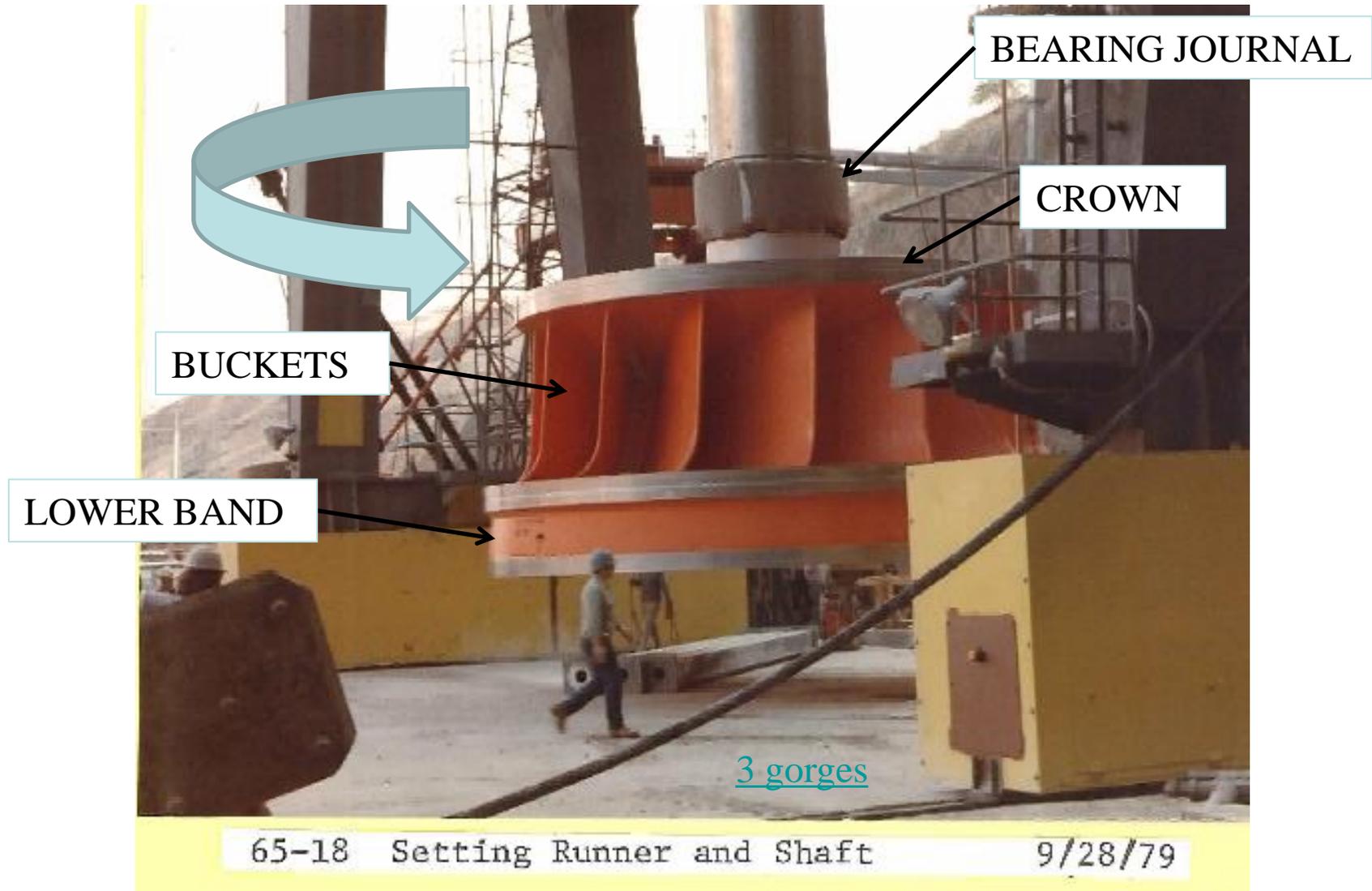


# Brownlee Plant History

- New 250MW Hitachi unit 5 added to plant in 1980



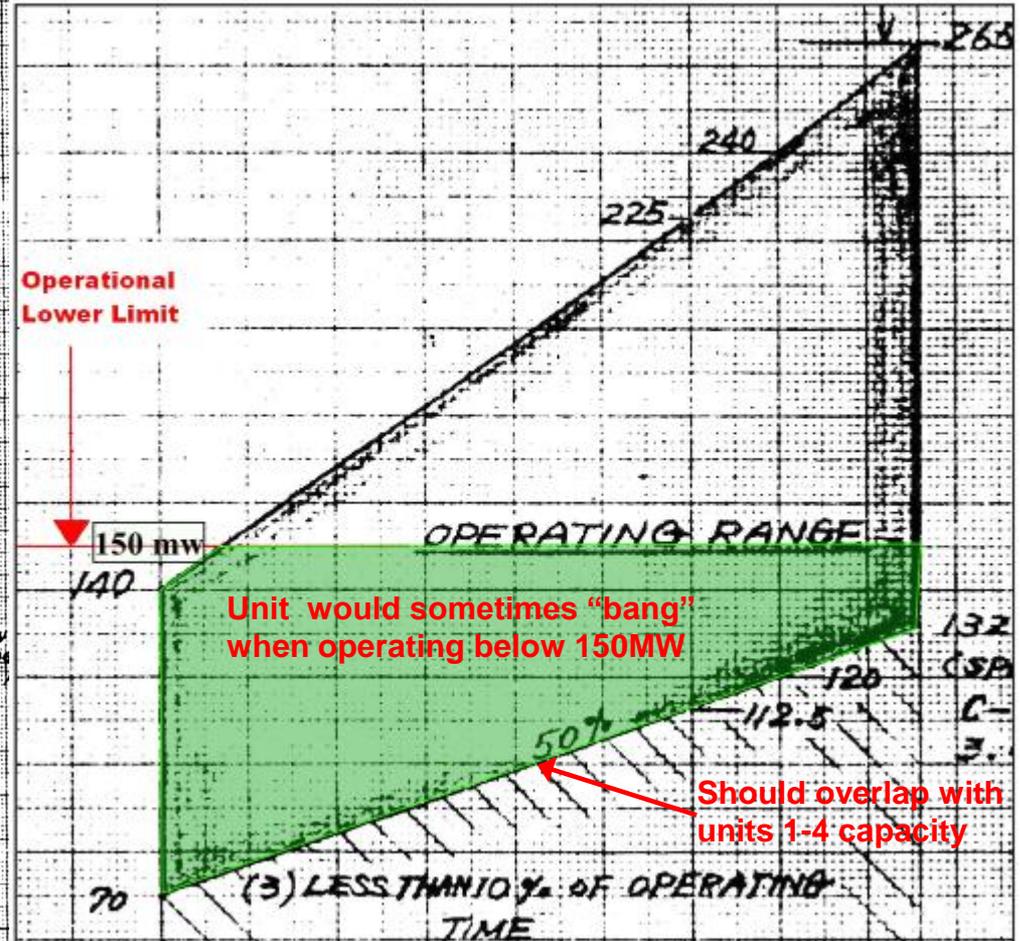
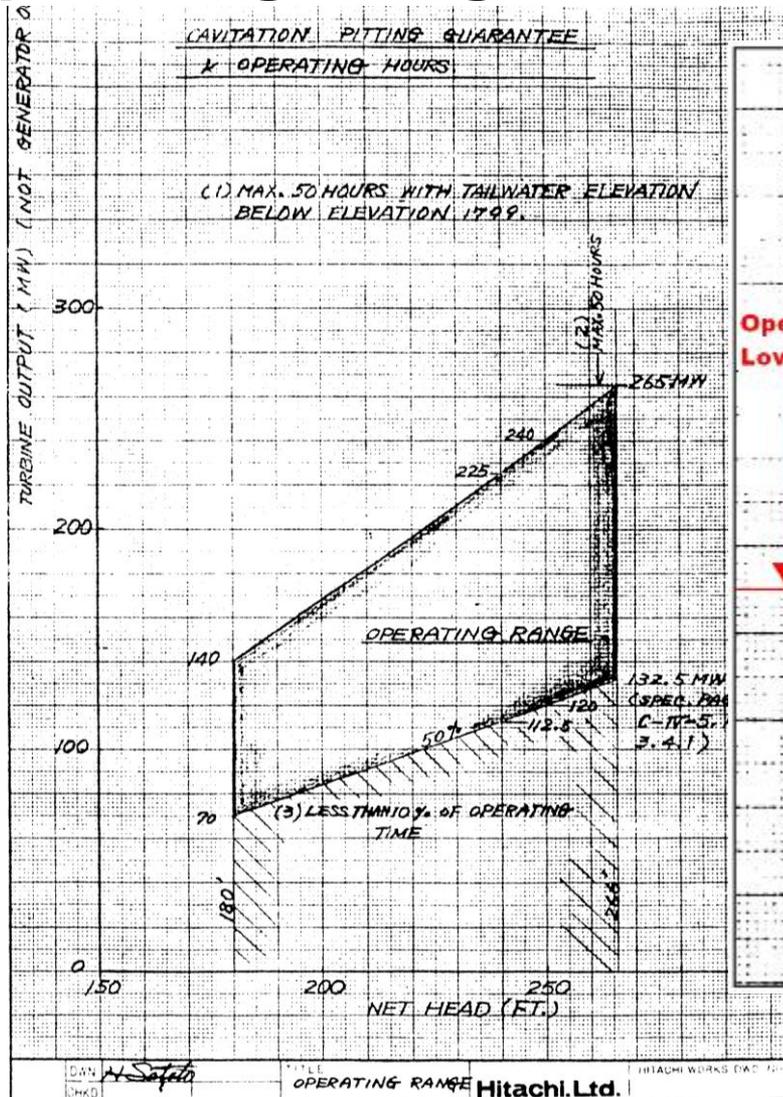
# Unit 5 Francis Turbine (Runner)



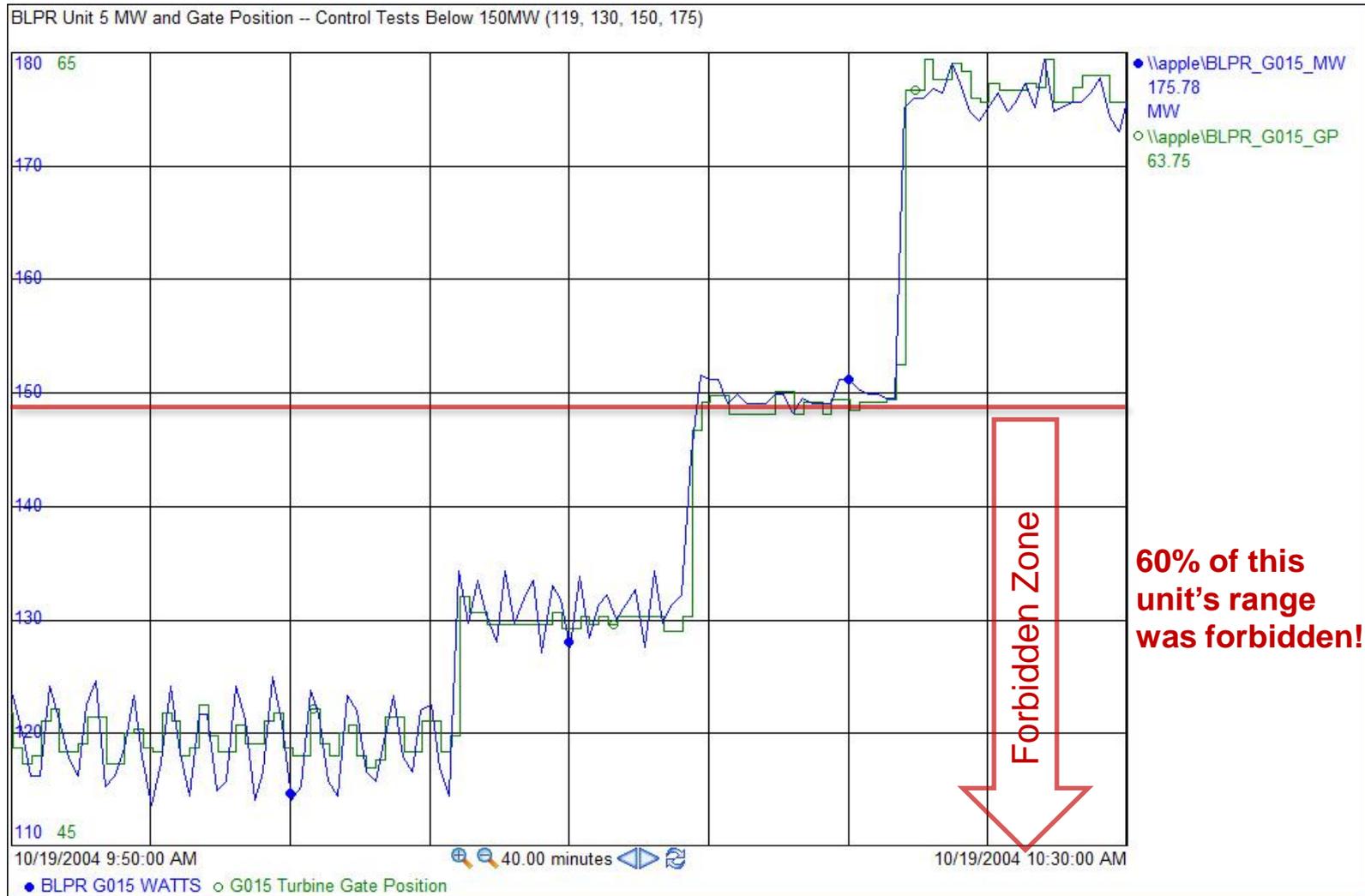
# Unit 5 Rough Zone History (1) ...

- A severe rough zone was identified during commissioning in **1980**
  - avoid that zone (below 50% gate), why operate off peak efficiency?
- Higher rough zone limits noticed during **1997** flood control draft
  - probably an issue with the vent valve operated by gate position
- MW oscillations noticed during AGC testing in **2004**
  - tested AVR and PSS to rule out control or electrical causes
  - focused attention on hydraulics (turbine?)
  - AGC implemented with forbidden zone below 150MW
- Generation dispatch requested a study to expand unit 5's operating zone to help with regulating margin and unit swapping in **2006**
  - of course once they got used to having it on AGC control....
  - need became urgent as we tried to integrate more wind generation

# Brownlee Unit 5 Guaranteed Operating Range



# Brownlee Unit 5 MW Oscillations below 150MW

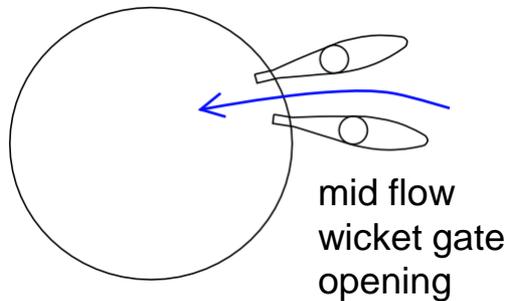
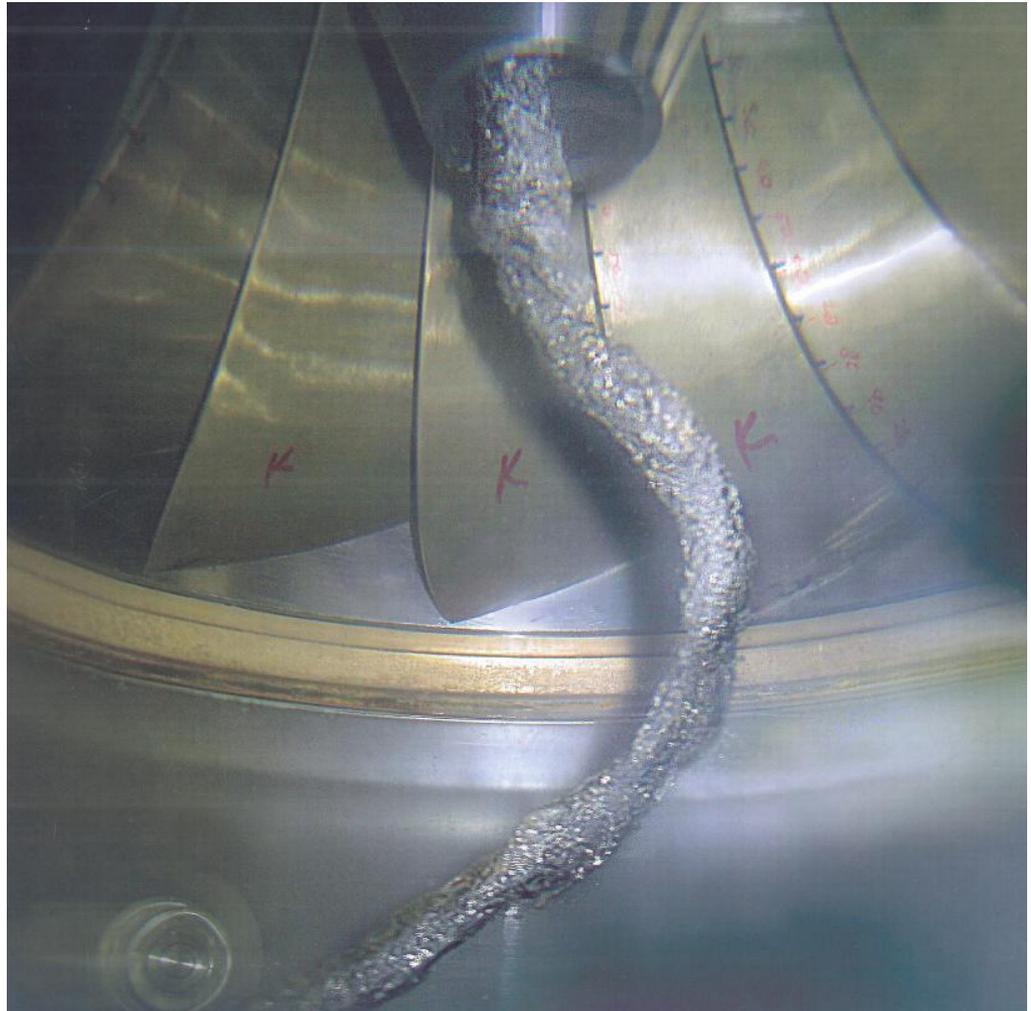


## ...Unit 5 Rough Zone History (2)

- In **2008** a consultant was hired for the rough zone study
  - **Rheingan's vortexing** in mid-gate regions
  - risk of vortex collapse and “bang” only happens in upper vortex zone 2
  - **blade cavitation** in lower-gate regions
  - rough zones vs. head water and gate position seemed well-behaved
  - discovered unused piping system for injecting compressed air into turbine
- Improved vent valve control could help in **vortexing** region
  - originally implemented with cam-driven switches in governor cabinet
  - use PLC to operate valves with both head-water and gate position (or MW)
  - use a **second** vent valve (normally used to vent depression air)
- Compressed air injection could help in the **cavitation** region
  - use the “**compressed air subsystem**” piping just discovered
  - used depression air tanks to inject pressurized air in front of the turbine blade
  - air-flow requirements estimated at **5000cfm**, 1000 hp, **750kW** electrical service

# Draft Tube Vortex (Rheingans)

- Swirl occurs below 50% wicket gate opening;
- Vortex rope spins at 20-30% of running speed;
- Needs lots of venting air to avoid collapse;
- Vortex collapse can cause large vibrations at Browlee 5 plant



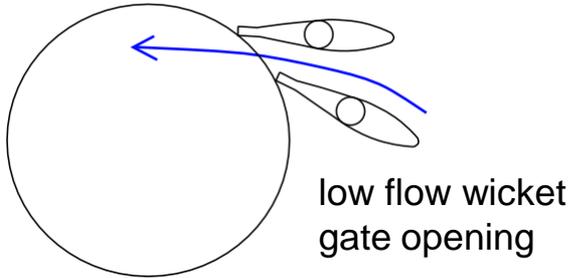
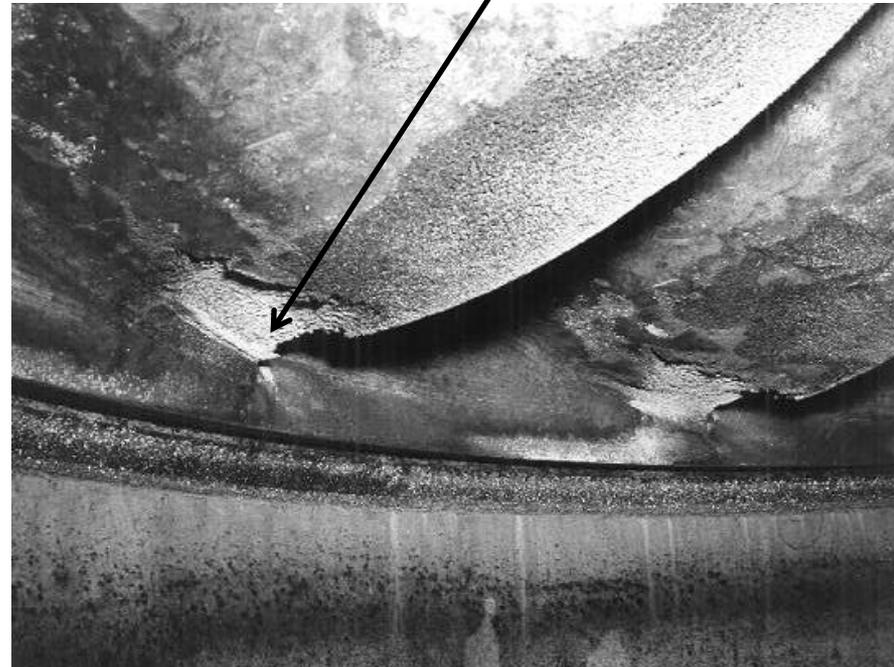
[Vortex Animation](#)

# Cavitation Damage



INLET EDGE CAVITATION DAMAGE

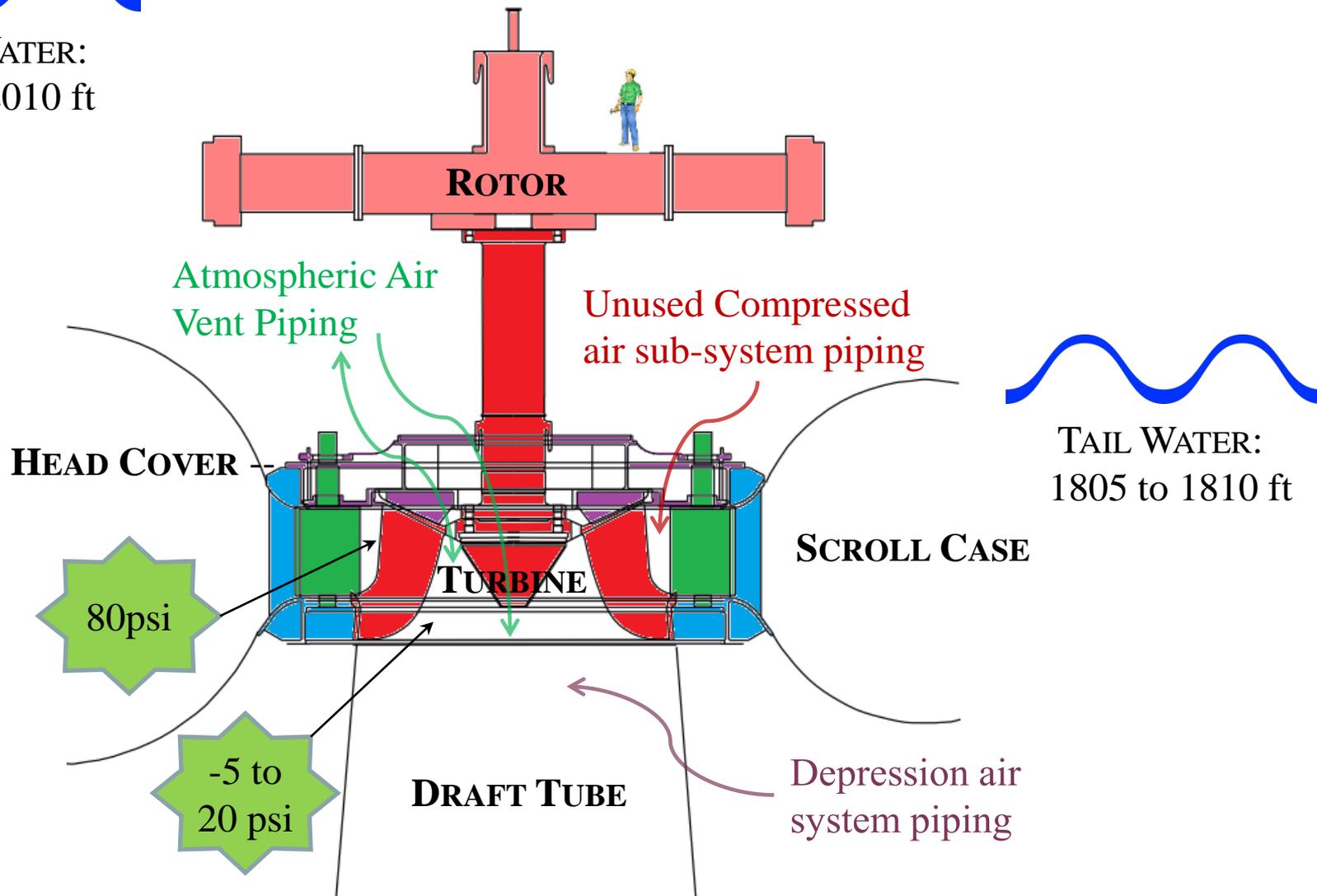
EXIT EDGE CAVITATION DAMAGE



# Brownlee Unit 5 Turbine Air



HEAD WATER:  
2077 to 2010 ft



TAIL WATER:  
1805 to 1810 ft

# Temporary Air Injection for 2008 Testing



used air stored in depression air tanks for short duration smoothing air injection tests

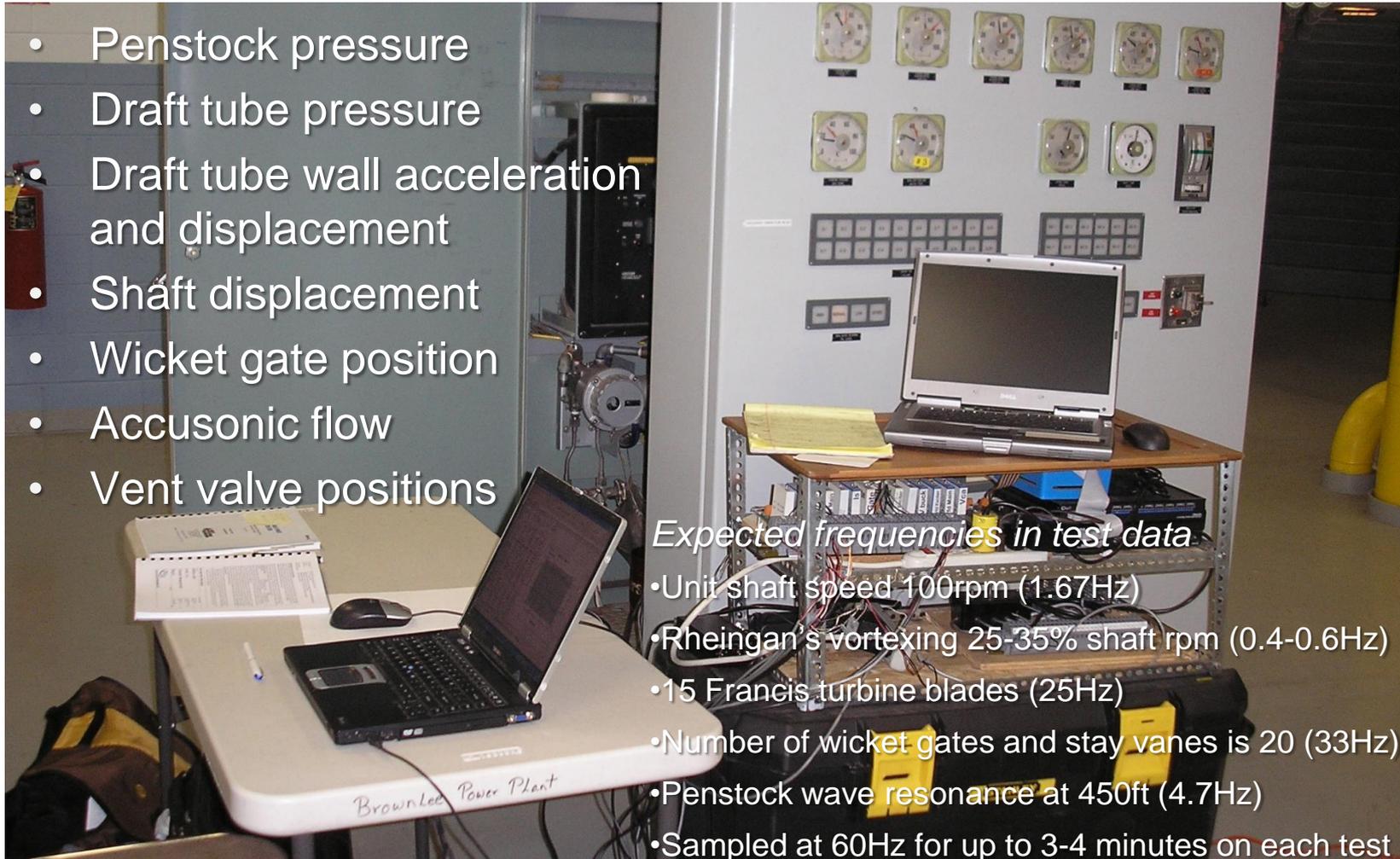


# 2008 Rough Zone Study Test Equipment Set-up

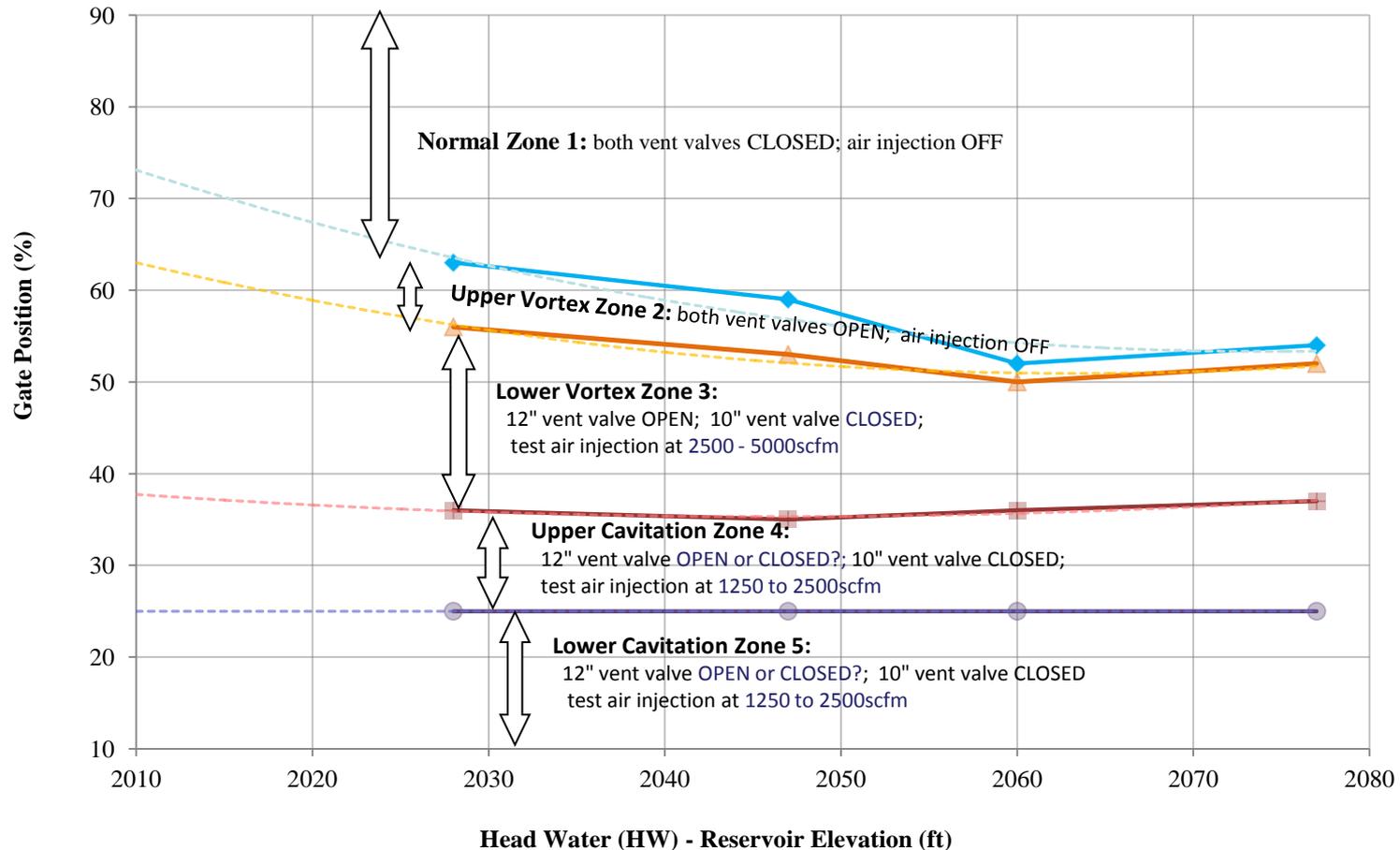
- Penstock pressure
- Draft tube pressure
- Draft tube wall acceleration and displacement
- Shaft displacement
- Wicket gate position
- Accusonic flow
- Vent valve positions

## *Expected frequencies in test data*

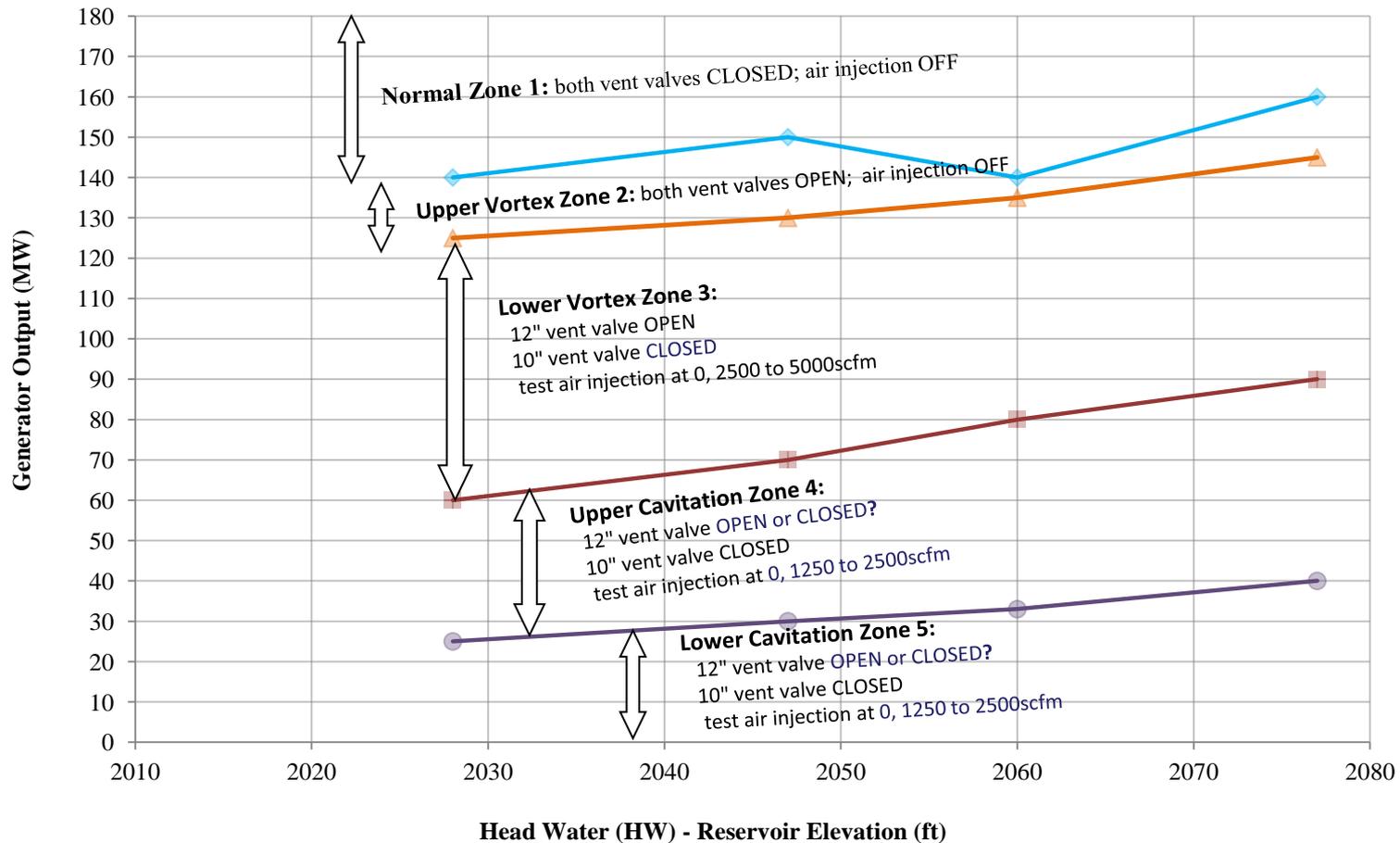
- Unit shaft speed 100rpm (1.67Hz)
- Rheingan's vortexing 25-35% shaft rpm (0.4-0.6Hz)
- 15 Francis turbine blades (25Hz)
- Number of wicket gates and stay vanes is 20 (33Hz)
- Penstock wave resonance at 450ft (4.7Hz)
- Sampled at 60Hz for up to 3-4 minutes on each test



# 2008 Operating Zone Map - Gate Position (non-linear)



# 2008 Operating Zone Map - MW (more linear)



## ...Unit 5 Rough Zone History (3)

- In **2010** we decided to install the air compressor system
  - least expensive alternative for gaining regulating margin on our system
  - compressor system losses and lost unit efficiency still less than depressing unit
  - needed the down regulation with increasing wind generation on our system
- In **2011** we installed a vibration monitoring system, and started...
  - history of damaged air piping on this unit if operated in rough zone
  - generations of plant operators learning not to operate the unit below 150MW
  - severe “banging” and some head-cover movement if vortex collapses in zone 2
  - Sayano-Shushenskaya hydro plant accident in Russia
- ...a test program to determine control requirements in the rough zones
  - map out unit 5’s operating zones with smoothing air mitigation
  - extend operating zones as far as possible
  - determine vibration alarm levels to protect machine

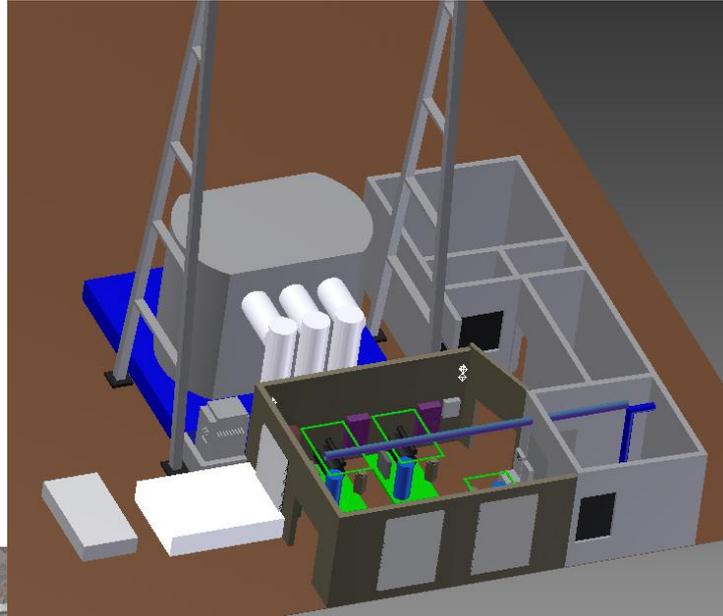
# Compressor Building before...

## Challenges:

- Physical Location
- Supply Power
- Control Integration



# Compressor Building after...



# Inside Compressor Building

## *System specifications:*

- Four 300-hp rotary screw compressors (1 VFD)
- 1,250 scfm of air delivery each
- 50-100 psi delivery pressure (variable)
- Sequencer/controller interfaced with plant PLC



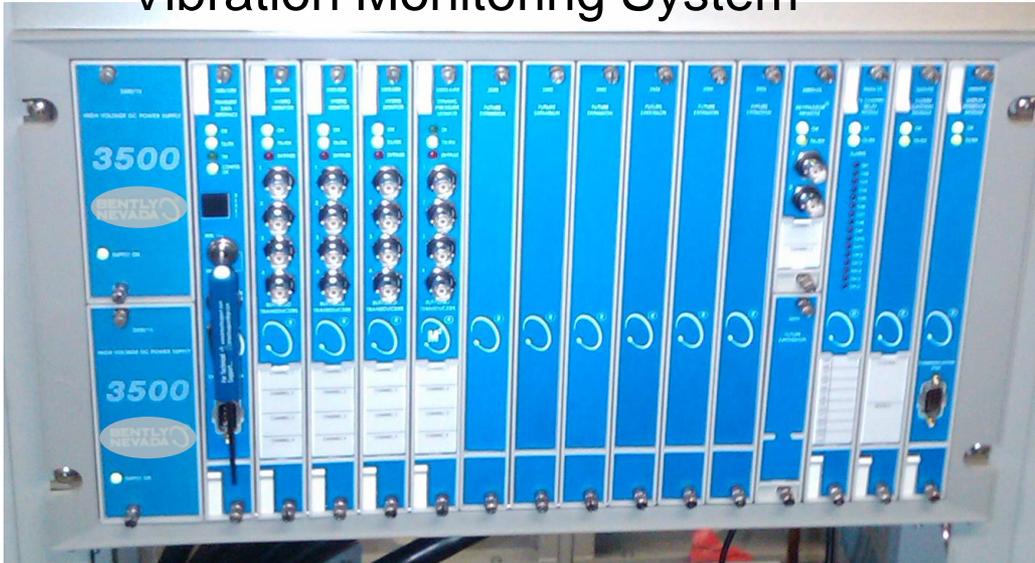
# Power Taps into Iso-Phase Bus

- used two spare local service transformers
- current-limiting fuses needed for fault duty
- very tight fit, but sufficient cooling air flow



# 2011 Operating Zone Mapping Test Equipment Set-up

## Vibration Monitoring System



## Fast sampled data:

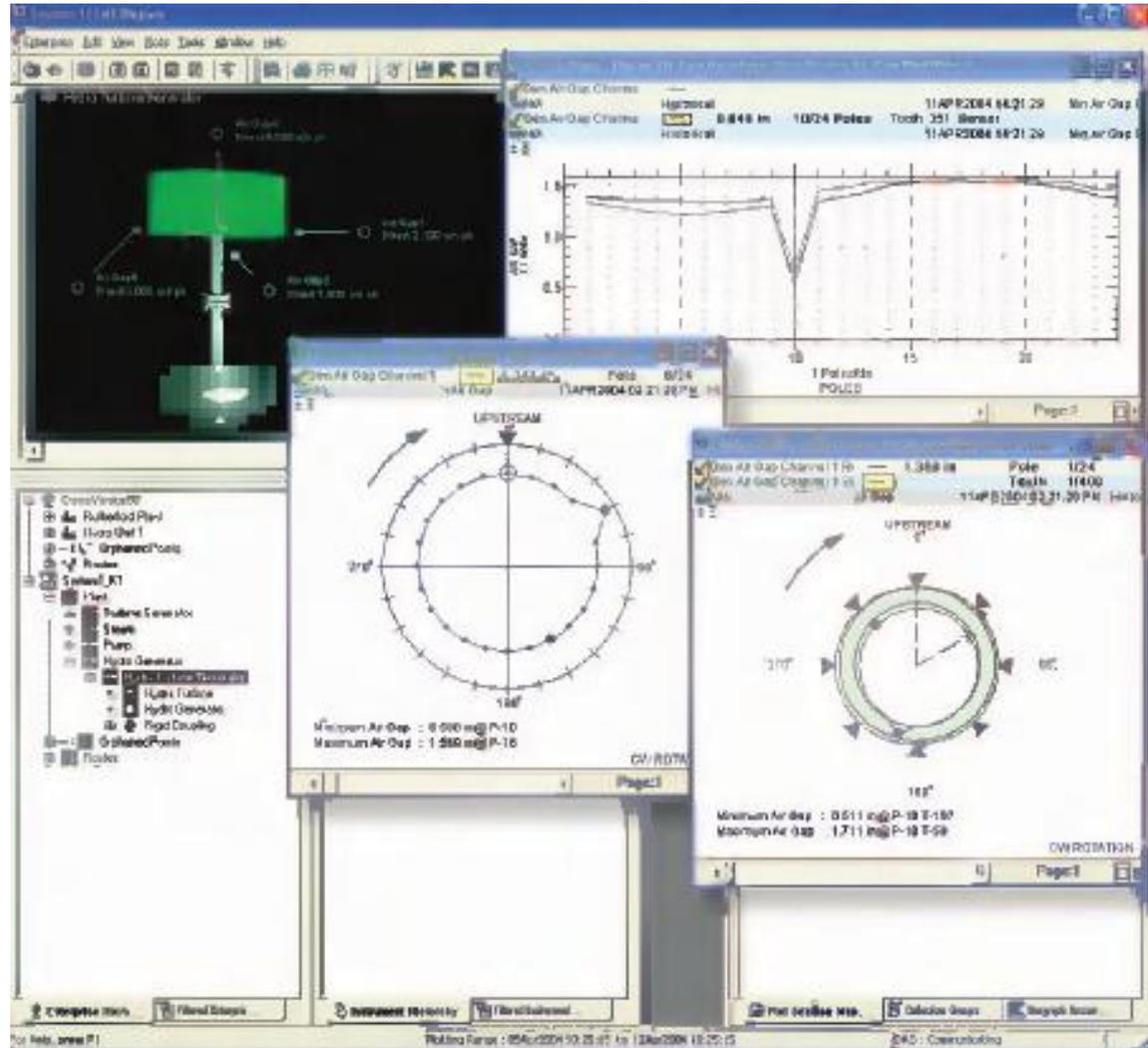
- Penstock pressure
- Draft tube pressure
- Draft tube acceleration
- Head-cover acceleration
- UGB vibration
- TGB vibration
- Thrust on shaft

## Slower sampled data:

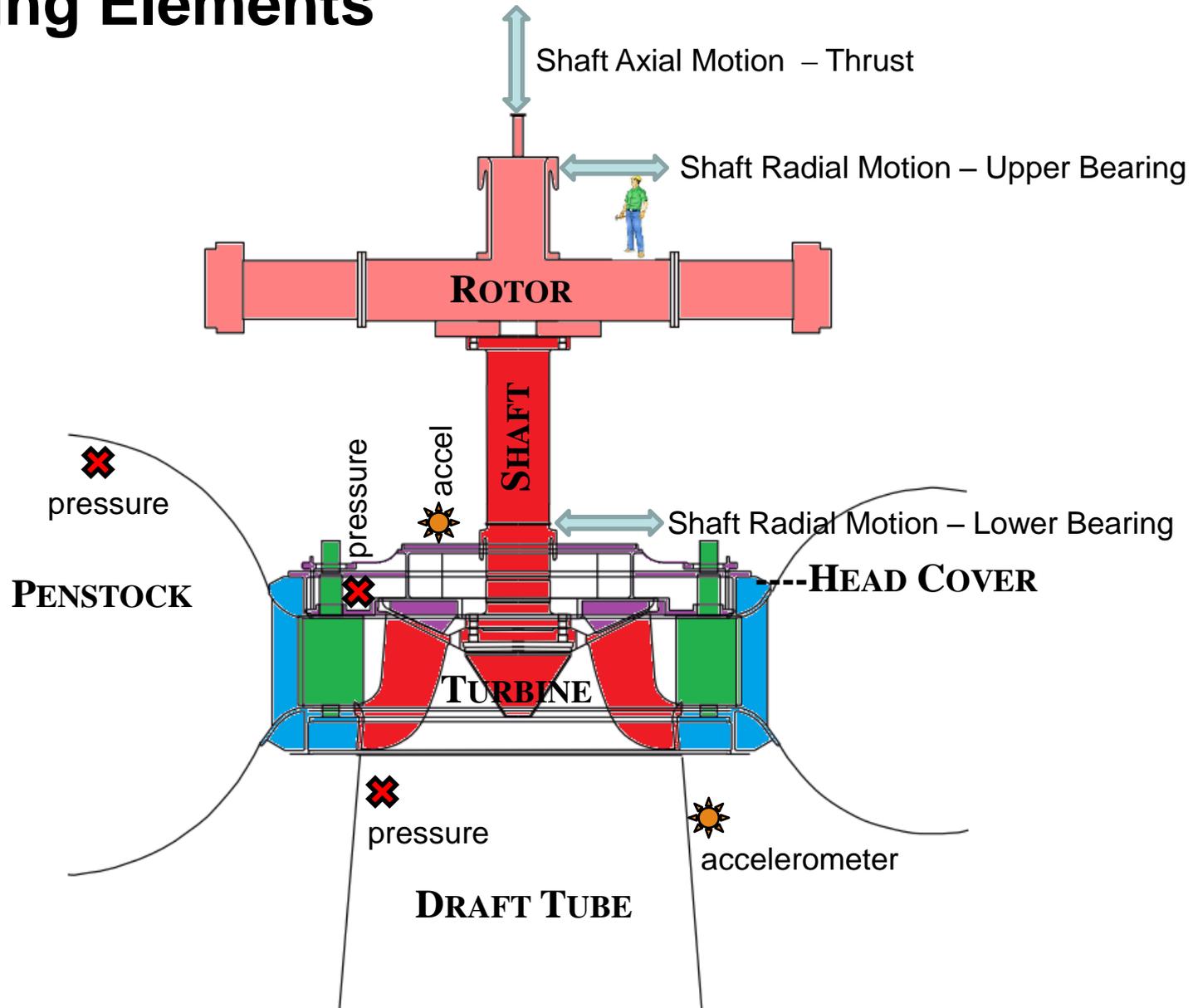
- MW output
- Wicket gate position
- Accusonic flow
- Air valve positions
- Air flow and pressures

# Data Collection & Analysis Software

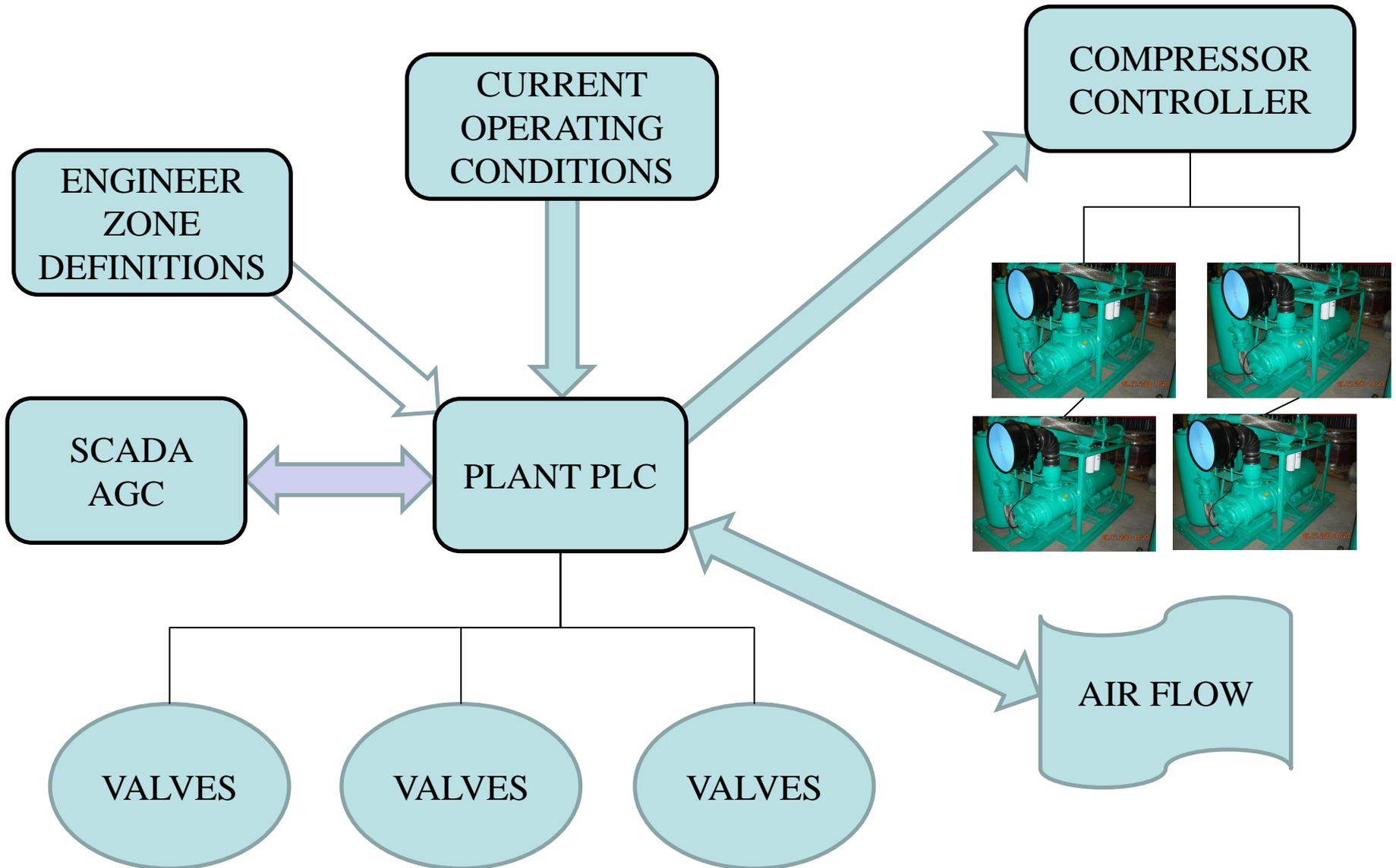
- Permanent data collection system
- Additional machinery diagnostic capabilities



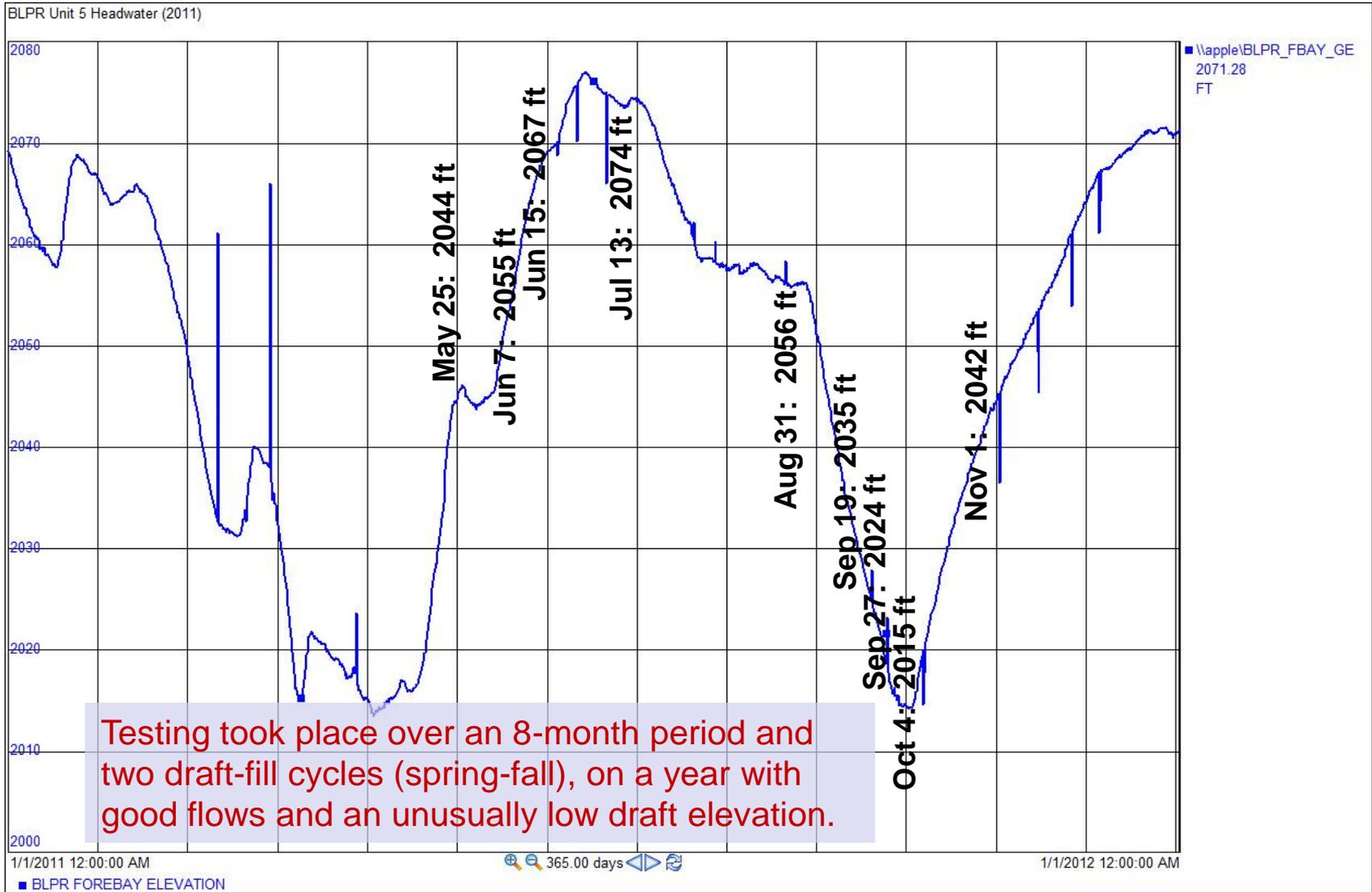
# 2011 Sensor Locations- Rotating Elements



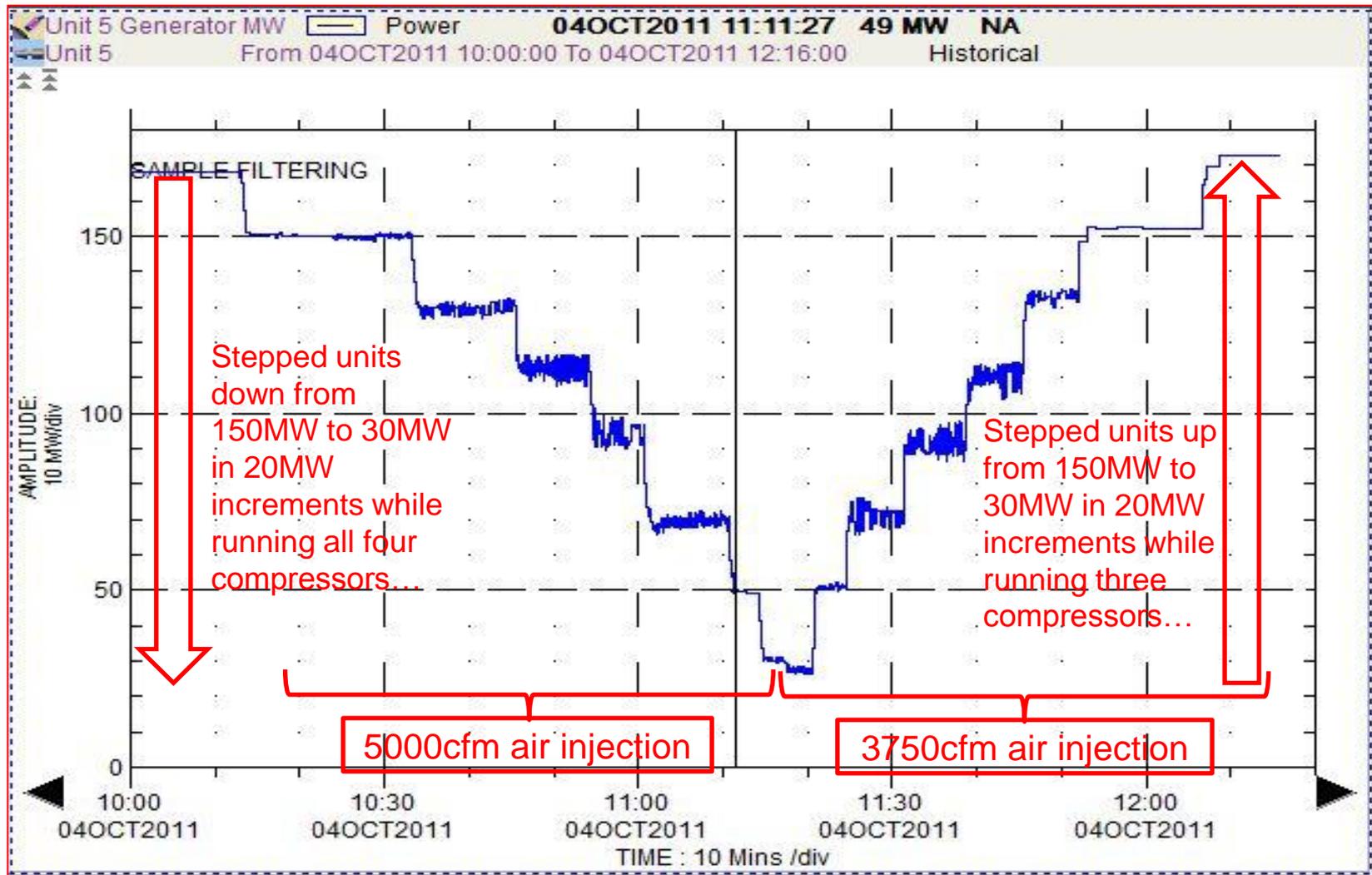
# Control System Overview



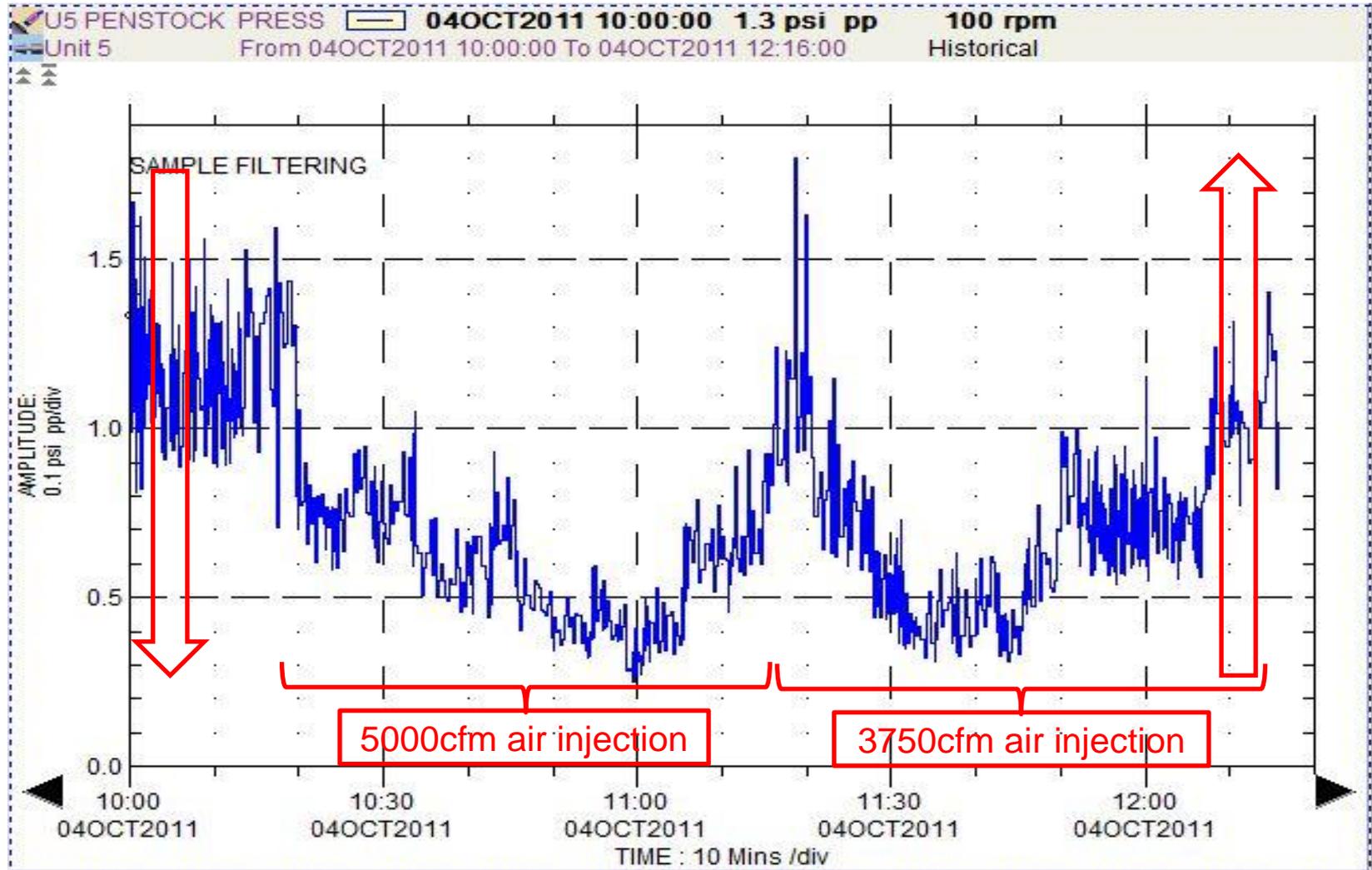
# Brownlee Headwater 2011 Test Opportunities



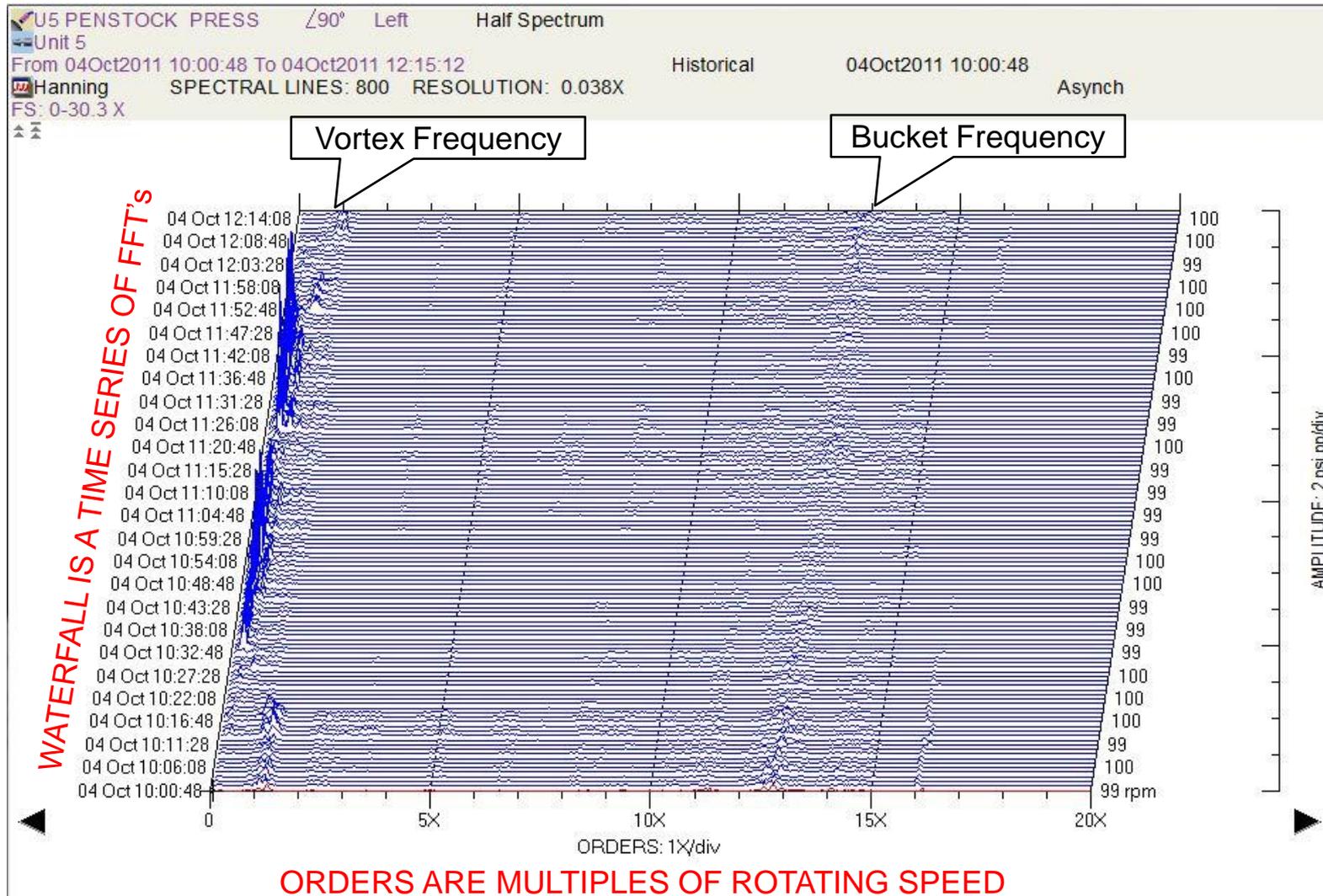
# MW variation at 2015ft



# Penstock Press at 2015ft

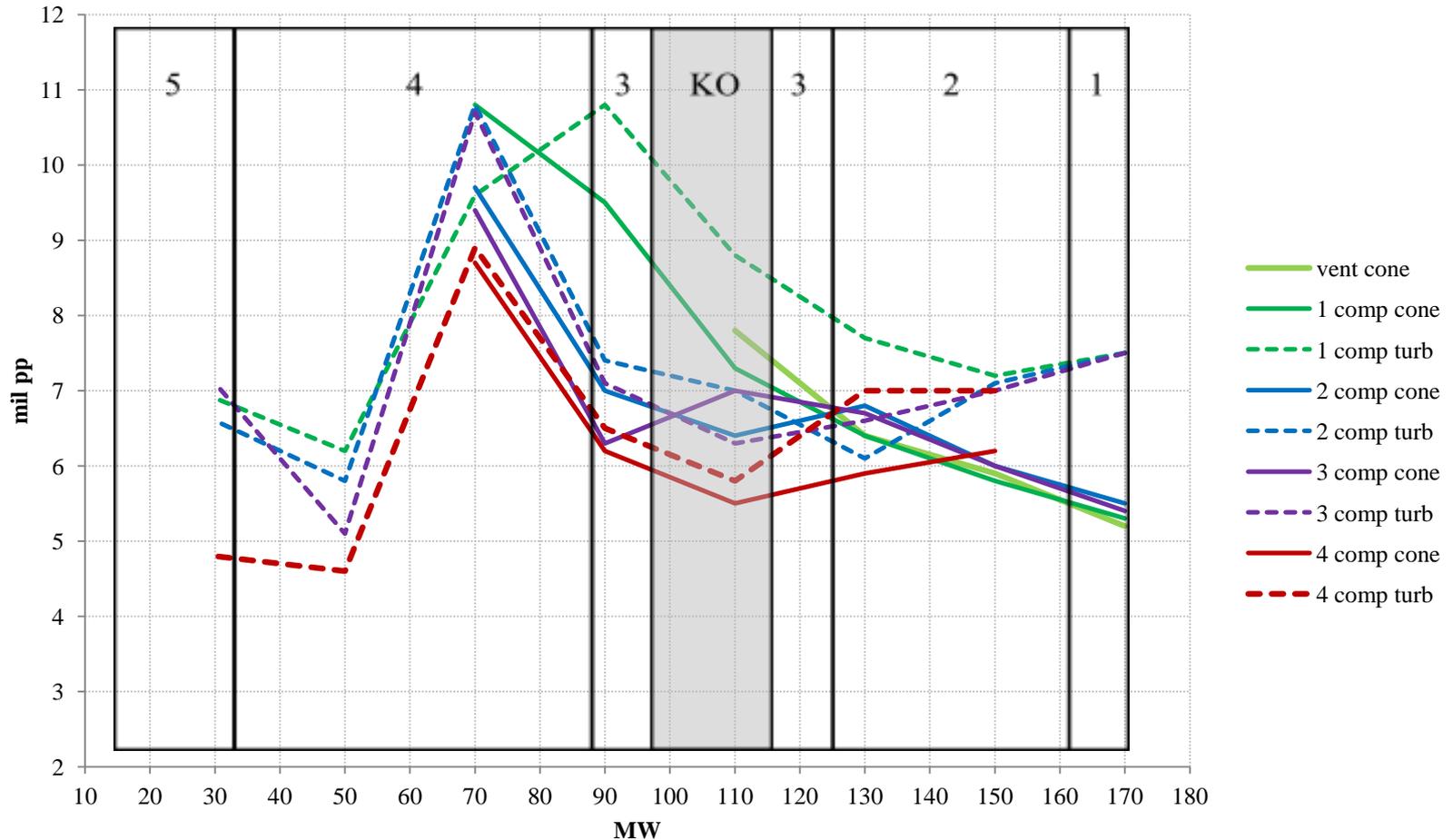


# Penstock Press FFTs at 2015ft



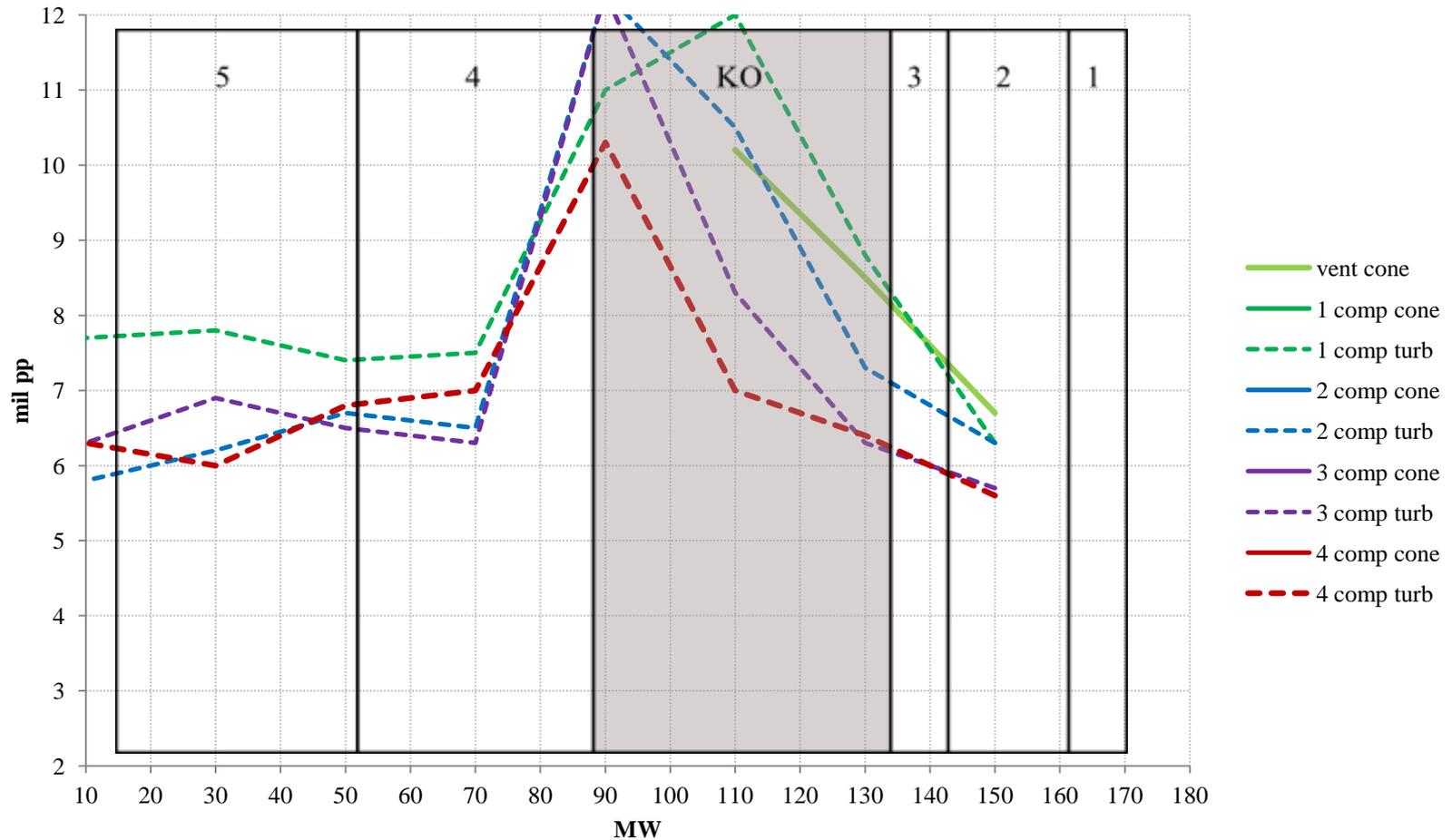
# Turbine Guide Bearing Max Vibration

HW = 2015ft TGB Upstream Vibration

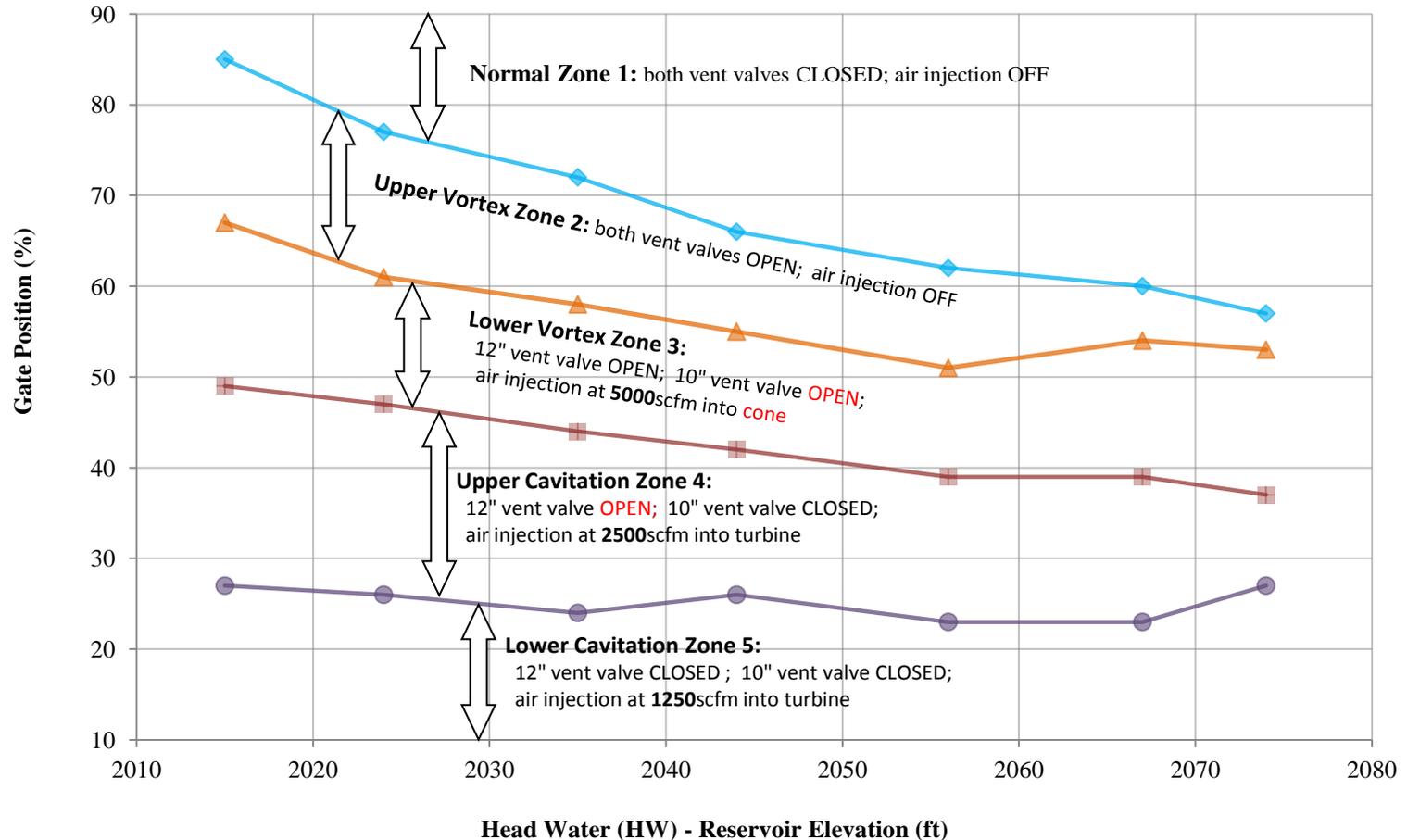


# Turbine Guide Bearing Max Vibration

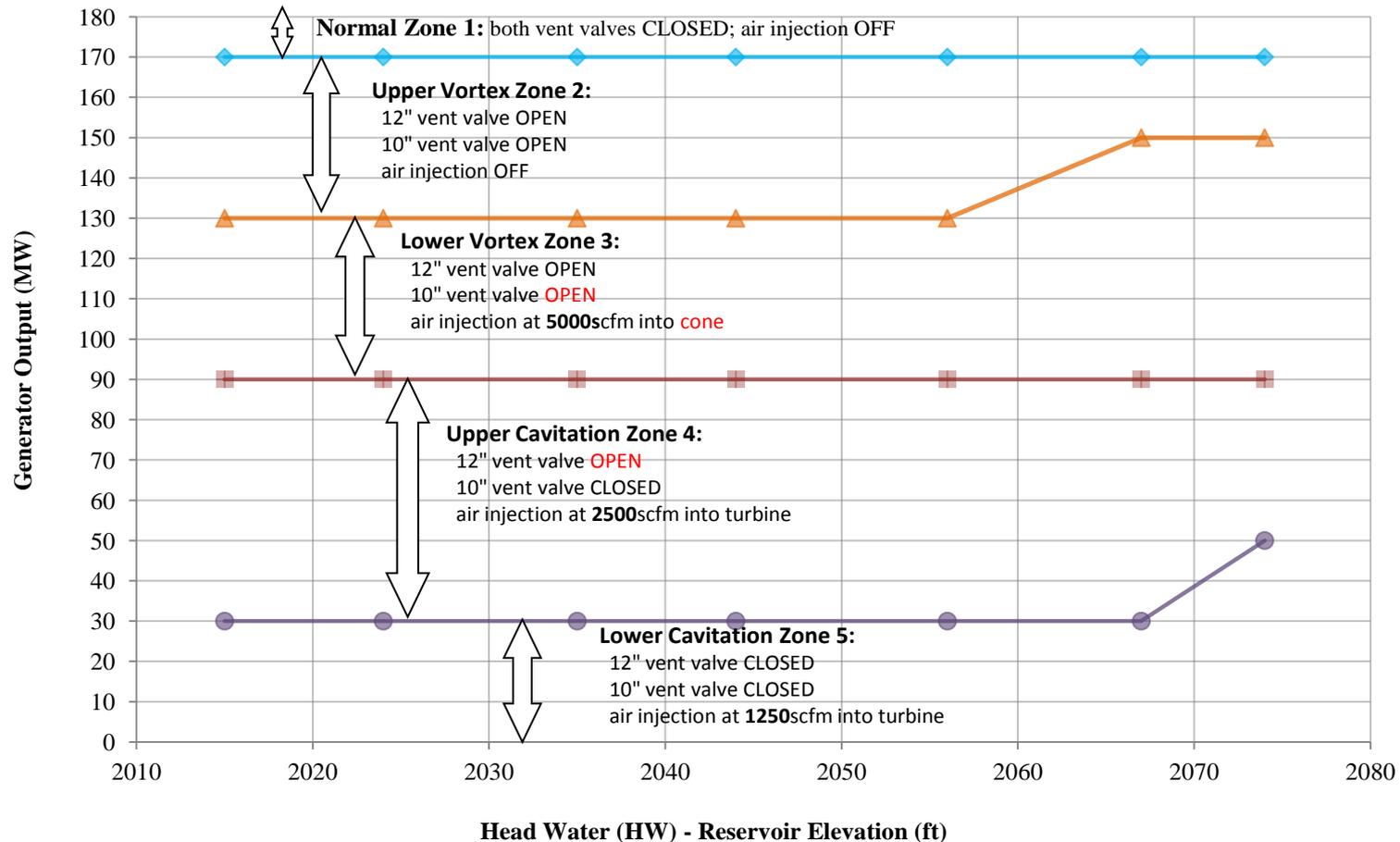
HW = 2074ft TGB Upstream Vibration



# 2011 Operating Zone Map – Gate Position (open valves earlier at low head)



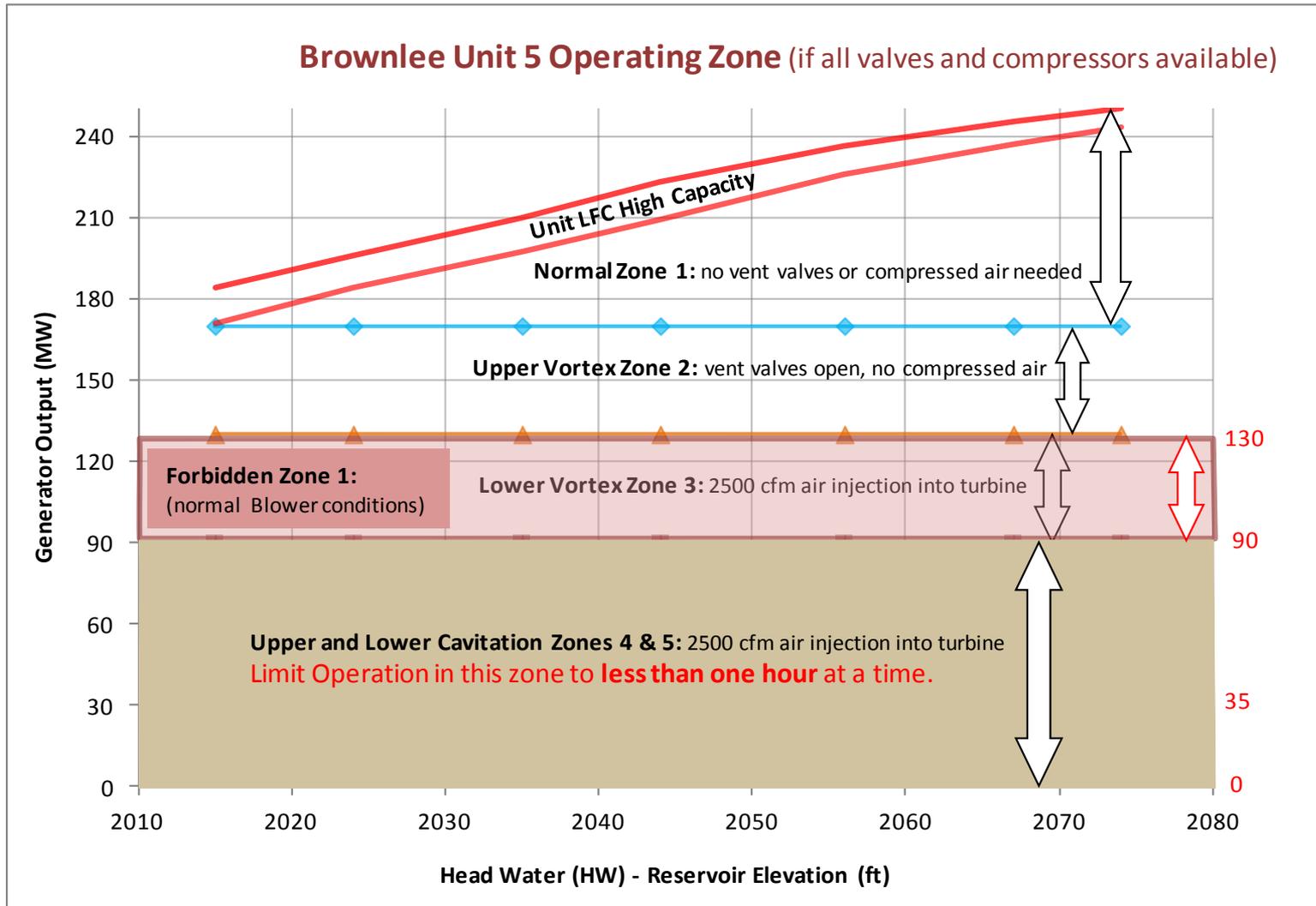
# 2011 Operating Zone Map – MW (surprisingly linear – easier PLC program)



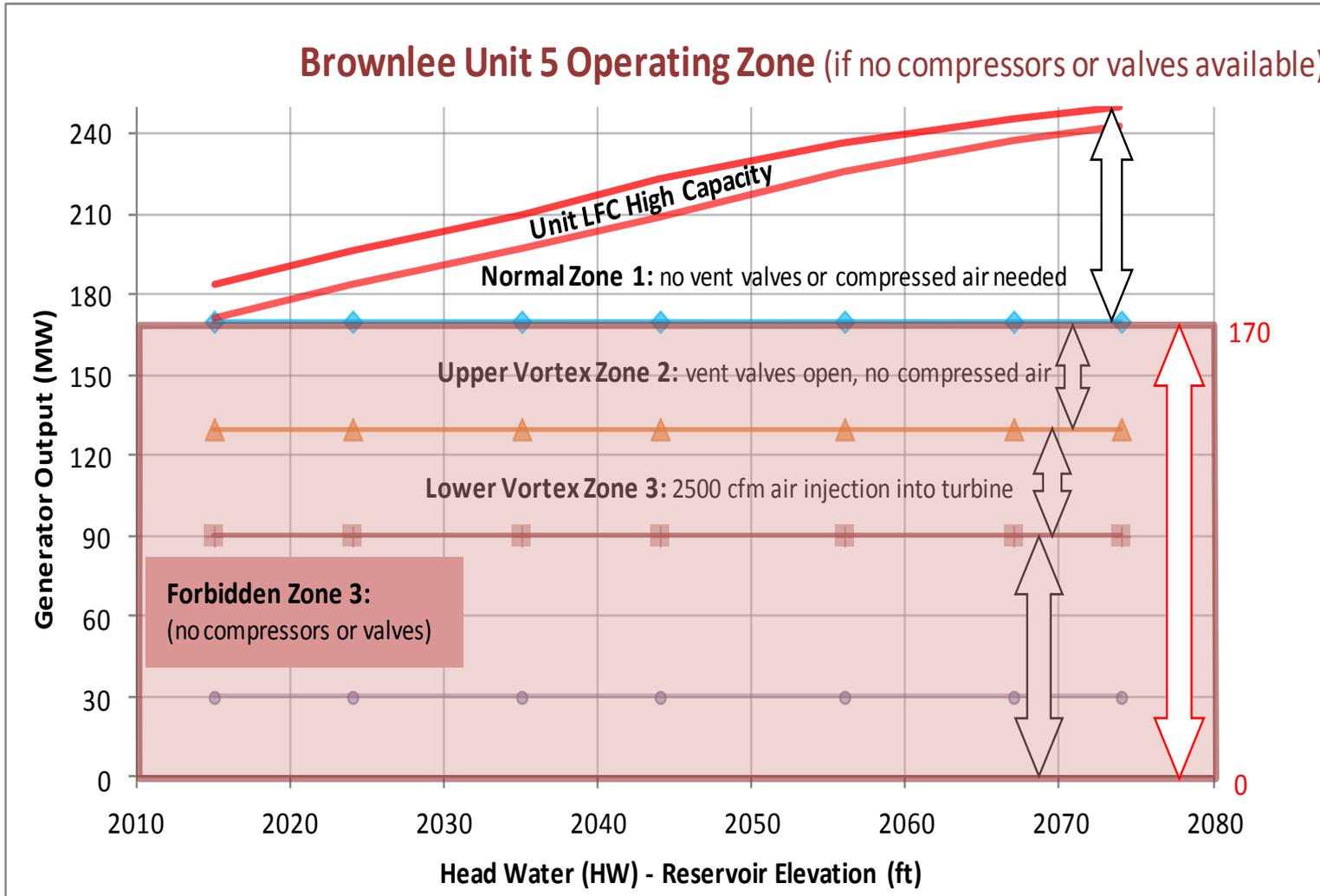
## ...Unit 5 Rough Zone History (4)

- Implemented compressor and vent valve controls in **2012**
  - forbidden zone limits depend on valve and compressor control conditions
  - PLC telemeters limits to AGC, which honors the forbidden zone
  - PLC moves unit out of rough zone and turns off AGC if high vibration detected
  - continuous vibration monitoring, event logging, and protection
- Using the system during normal operations on **2013**
  - with a “dynamic” forbidden zone limits from plant PLC to AGC

# 2013 Operating Zone Map – MW (for dispatch and operations)



# 2013 Operating Zone Map – MW (for dispatch and operations)



# Lessons Learned (1) ...

- Francis Turbine vortex rough zones are worse in some turbines
  - history of “bangs” in Brownlee unit 5 plant, but no issues with units 1-4
  - say Sayano-Shushenskaya?
- Detecting a vortex collapse can be tricky
  - random timing on a non-periodic signal
  - thrust forces cause mostly axial shaft and head-cover movement
  - know your zone boundaries
- Operating in the vortex rough zone presents risk but is possible
  - maximize air flow with venting and compressed air controls
  - or design controls to pass through zone quickly
- Operating in the cavitation rough zone is easier to do and less risky
  - need enough compressed air and controls to get it when and where it is needed
  - reduce risk with vibration monitoring
  - need to evaluate long-term effects on turbine blades or buckets

## ...Lessons Learned (2)

- Big compressors can be expensive as a retro-fit
  - large air volume requirements, tight fit in old plant
  - do a more detailed design and budget up-front
- Testing needed to define rough zone boundaries can take years
  - need to catch a range of reservoir levels and flows
  - dependent on dispatch and crew availability
  - can't schedule testing ahead of time for duration of project
- Need long term monitoring, maintenance, and engineering support
  - control performance monitoring and tuning
  - somebody needs to look at all that vibration data
  - challenge to keep resources focused

# Questions?

John Heiselmann  
Principle Engineer  
[jheiselmann@idahopower.com](mailto:jheiselmann@idahopower.com)  
208-388-6077